

In the Claims:

Claims 1 to 14 (Canceled).

1 15. (Currently amended) A method for assisting the driver of a  
2 vehicle (10) when performing a driving maneuver formed by  
3 a parking or shunting maneuver, a reference trajectory (16)  
4 corresponding to the driving maneuver being determined,  
5 along which the vehicle ~~[[19]]~~ (10) is to be moved, and  
6 the steering wheel position to be set in each case and  
7 controlling the vehicle (10) along the reference trajectory  
8 ~~[[16, 19]]~~ (16) being indicated to the driver during the  
9 driving maneuver, the vehicle longitudinal speed (v) being  
10 influenced independently of the driver in the event of a  
11 steering angle deviation ( $d_{lw}$ ) between the actual steering  
12 angle ( $\delta_{act}$ ) actually set by the driver and the desired  
13 steering angle ( $\delta_{des}$ ) corresponding to the requested steering  
14 wheel position, characterized in that the vehicle  
15 longitudinal speed is influenced on the basis of the  
16 magnitude of the steering angle deviation ( $d_{lw}$ ) in such a  
17 way that ~~[[the]]~~ a greater ~~[[the]]~~ vehicle retardation is  
18 carried out, the greater the magnitude of the steering  
19 angle deviation ( $d_{lw}$ ) is.

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1 16. (Previously presented) The method as claimed in claim 15,  
2 characterized in that, during the driving maneuver,  
3 depending on the current vehicle position ( $x_{F,act}/y_{F,act}/\psi_{F,act}$ ),  
4 a steering angle tolerance band ( $\delta_{min}$  to  $\delta_{max}$ ) which  
5 determines the permissible steering angle is determined and  
6 the influence on the vehicle longitudinal speed ( $v$ ) depends  
7 on the tolerance margin ( $\delta_{des} - \delta_{min}$  or  $\delta_{max} - \delta_{des}$ ) between the  
8 desired steering angle ( $\delta_{des}$ ) and the tolerance band limits  
9 ( $\delta_{min}$  or  $\delta_{max}$ ).

1 17. (Previously presented) The method as claimed in claim 16,  
2 characterized in that, in order to determine the steering  
3 angle tolerance band, a rotational angle tolerance band is  
4 determined, the actual rotational angle ( $\psi_{F,act}$ ) between the  
5 vehicle longitudinal axis (71) and a coordinate axis ( $y$ ) of  
6 a stationary coordinate system (22) being enlarged or  
7 reduced until it is just still possible to determine a  
8 trajectory to the target position (17).

1 18. (Previously presented) The method as claimed in claim 16,  
2 characterized in that the vehicle longitudinal speed ( $v$ ) is  
3 chosen to be lower, the smaller the magnitude of the  
4 tolerance margin ( $\delta_{des} - \delta_{min}$  or  $\delta_{max} - \delta_{des}$ ).

1 19. (Previously presented) The method as claimed in claim 15,  
2 characterized in that the vehicle longitudinal speed ( $v$ ) is  
3 chosen to be lower, the greater the magnitude of the  
4 steering angle deviation ( $d_{LW}$ ).

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1     **20.** (Currently amended) The method Method as claimed in claim  
2           15, characterized in that the vehicle longitudinal speed is  
3           influenced by means of speed regulation.

1     **21.** (Previously presented) The method as claimed in claim 15,  
2           characterized in that the vehicle (10) is retarded down to  
3           a standstill and is kept at a standstill as long as, on the  
4           basis of the existing steering angle deviation ( $d_{LW}$ ), the  
5           vehicle (10) would assume a vehicle position during onward  
6           travel from which the target position (17) can no longer be  
7           reached without a shunting interruption to the driving  
8           maneuver.

1     **22.** (Previously presented) The method as claimed in claim 21,  
2           characterized in that the vehicle (10) is accelerated again  
3           independently of the driver if the driver sets a steering  
4           wheel position which leads to a permissible steering angle  
5           deviation ( $d_{LW}$ ).

1     **23.** (Previously presented) The method as claimed in claim 15,  
2           characterized in that the steering wheel position to be set  
3           is indicated by means for acoustic driver information  
4           and/or means for optical driver information (13) and/or  
5           means for tactile driver information (40 and 41).

1 24. (Currently amended) The method as claimed in claim 23,  
2 characterized ~~[[is]]~~ in that the means for tactile driver  
3 information (40 and 41) have means for changing the  
4 steering wheel torque to be applied by the driver.

1 25. (Previously presented) The method as claimed in claim 15,  
2 characterized in that the driving maneuver is a parking  
3 maneuver and the reference trajectory (16) indicates the  
4 ideal route from the actual vehicle position  
5 ( $x_{F,act}/y_{F,act}/\psi_{F,act}$ ) into the parking position (17).

1 26. (Currently amended) The method as claimed in claim 15,  
2 characterized in that, in the case of a vehicle (10) in  
3 trailer operation, each vehicle position along the actual  
4 reference trajectory ~~[[ $\psi_{F,act}$ ]]~~ (16) is assigned a desired  
5 trailer angle ( $\beta_{des}$ ) between the vehicle longitudinal axis  
6 (71) and the trailer longitudinal axis (72), and in that  
7 the actual trailer angle ( $\beta_{act}$ ) is determined and compared  
8 with the corresponding desired trailer angle ( $\beta_{des}$ ), the  
9 vehicle longitudinal speed ( $v$ ) being influenced  
10 independently of the driver in the event of an angular  
11 deviation between desired trailer angle ( $\beta_{des}$ ) and actual  
12 trailer angle ( $\beta_{act}$ ).

1 27. (Currently amended) A device for implementing a method for  
2 assisting ~~[[the]]~~ a driver of a vehicle when performing a  
3 driving maneuver formed by a parking or shunting maneuver,  
4 ~~maneuver as claimed in claim 15,~~ having means (12) for

5 determining a reference trajectory (16) along which the  
6 vehicle (10) is to be moved corresponding to the driving  
7 maneuver, ~~[[and]]~~ means (13; 40 and 41) for indicating the  
8 steering wheel position to be set by the driver and  
9 controlling the vehicle (10) along the reference trajectory  
10 ~~[[+19],]~~ (16), an evaluation device, and retardation means  
11 (50) and/or forward drive means (51) for influencing the  
12 vehicle longitudinal speed (v), wherein the being  
13 influenced by retardation means (50) and/or forward drive  
14 means (51) ~~[[that]]~~ can be activated independently of the  
15 driver if a steering angle deviation ( $d_{LW}$ ) between the  
16 actual steering angle ( $\delta_{act}$ ) actually set by the driver and  
17 the desired steering angle ( $\delta_{des}$ ) corresponding to the  
18 requested steering wheel position is established in ~~[[an]]~~  
19 the evaluation device (12), characterized in that the  
20 vehicle longitudinal speed is influenced on the basis of  
21 the magnitude of the steering angle deviation ( $d_{LW}$ ) in such  
22 a way that ~~[[the]]~~ a greater ~~[[the]]~~ vehicle retardation is  
23 carried out, the greater the magnitude of the steering  
24 angle deviation ( $d_{LW}$ ) is.

1 28. (Currently amended) The device as claimed in claim 27,  
2 characterized in that means (12) are provided for  
3 determining the desired trailer angle ( $\beta_{des}$ ) between the  
4 vehicle longitudinal axis (71) and a ~~[[the]]~~ trailer  
5 longitudinal axis (70) of a trailer being towed by the  
6 vehicle, and means for determining the actual trailer angle  
7 ( $\beta_{act}$ ), in that the evaluation device (12) compares the

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8        desired trailer angle ( $\beta_{des}$ ) and the actual trailer angle  
9        ( $\beta_{act}$ ), and in that the retardation means (50) and/or forward  
10       drive means (51) of the vehicle (10) are activated in the  
11       event of an angular deviation being established between the  
12       desired trailer angle ( $\beta_{des}$ ) and the actual trailer  
13       angle ( $\beta_{act}$ ).

[RESPONSE CONTINUES ON NEXT PAGE]

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